# **Magneto-Electro-Thermo-Mechanical Response of a Multiferroic Doubly-Curved Nano-Shell**

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#### **ABSTRACT**

Free vibration of a simply-supported magneto-electro-elastic doublycurved nano-shell is studied based on the first-order shear deformation theory in the presence of the rotary inertia effect. To model the electric and magnetic behaviors of the nano-shell, Gauss's laws for electrostatics and magneto statics are used. By using Navier's method, the partial differential equations of motion are reduced to a single ordinary differential equation. Then, an analytical relation is obtained for the natural frequency of magneto-electro-elastic doubly-curved nano-shell. Some examples are presented to validate the proposed model. Moreover, the effects of the electric and magnetic potentials, temperature rise, nonlocal parameter, parameters of Pasternak foundation, and the geometry of the nano-shell on the natural frequencies of magneto-electro-elastic doubly-curved nanoshells are investigated. It is found that natural frequency of magnetoelectro-elastic doubly-curved nano-shell decreases with increasing the temperature, increasing the electric potential, or decreasing the magnetic potential. © 2018 IAU, Arak Branch. All rights reserved.

**Keywords:** Magneto-electro-elastic; Nano-shell; Doubly-curved; Firstorder theory.

# **1 INTRODUCTION**

**M** AGNETO-ELECTRO-ELASTIC (MME) materials are smart materials exhibiting magneto-electric (ME) coupling which enables them to convert mechanical, electrical and magnetic energies to each other. (ME) coupling which enables them to convert mechanical, electrical and magnetic energies to each other. Understanding the vibration behavior of nano-structures is the key step in designing nano-sized devices like sensors. So, it is important to study the vibration response of MEE nano-structures before using them as sensors, actuators, energy harvesters, etc.

Arash and Wang [1] introduced and reviewed several nonlocal continuum models to model carbon nano-tubes and graphene sheets. Pradhan and Kumar [2] and Babaei and Shahidi [3], respectively, used differential quadrature (DQ) and Galerkin methods to determine the vibration response of graphene sheets. Three-dimensional (3D) theory [4], Mindlin plate theory [5], higher-order shear deformation theory (HSDT) [6], and refined plate theory (RPT) [7] in conjunction with nonlocal continuum theory are used to study the vibration behavior of nano-plates. Analooei et al. [8] used finite strip method to study the vibration of isotropic and orthotropic nano-plates. Aksencer and Aydogdu [9] used Navier-type solution for vibration analysis of simply-supported nano-plate. They also used Levytype solution for vibration analysis of nano-plates with two opposite edges simply-supported and the others

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arbitrary. Tadi Beni et al. [10] and Rouhi et al. [11] presented shear deformable models for free vibration analysis of cylindrical nano-shells. Pouresmaeeli et al. [12] studied the vibration behavior of simply-supported viscoelastic nano-plate based on nonlocal Kirchhoff theory. They found that the frequency decreases with increasing the structural damping coefficient. Ghorbanpour Arani et al. [13] studied the free vibration of a magnetostrictive nanoplate based on Reddy's third-order shear deformation theory (TSDT) and Eringen's nonlocal continuum model. Vibration of nano-tube reinforced fluid-conveying micro-tubes supported by visco-Pasternak foundation has also been investigated by Arani et al. [14]. Liu et al. [15] and Ke et al. [16] investigated vibration of piezoelectric nanoplates based on nonlocal Kirchhoff and nonlocal Mindlin theories, respectively. Ke et al. [17] studied thermoelectro-mechanical vibration of piezoelectric cylindrical nano-shells using nonlocal Love's thin shell theory. Vaezi et al. [18] investigated the effects of electric and magnetic potentials on the free vibration of a MEE micro-beam based on Euler-Bernoulli beam theory. Amiri et al. [19] used a MEE beam model to study the vibration and instability of fluid conveying smart micro-tubes. Ebrahimi and Barati [20] examined vibrational behavior of a magneto-electro-viscoelastic nano-beam based on HSDT. Li et al. [21] studied the bending, buckling and free vibration of MEE nano-beams based on nonlocal Timoshenko theory. They determined the effects of electric and magnetic potentials on the vibration response of these nano-beams. Nonlocal TSDT [22] and nonlocal Timoshenko theory [23] have been used by Ansari et al. to study the nonlinear forced vibration of a MEE nano-beam. Ke and Wang [24] used nonlocal Timoshenko theory in conjunction with DQ method to obtain the free vibration response of a MEE nano-beam. Ke et al. [25] studied the free vibration of a MEE nano-plate based on the nonlocal Kirchhoff plate. They found that the natural frequency of the MEE nano-plate is sensitive to mechanical, electrical and magnetic loadings, while it is insensitive to the thermal loading. By considering surface and nonlocal effects, Wang et al. [26] introduced a two-dimensional theory to study the response of MEE nano-plates. Li et al. [27] investigated buckling and free vibration of a MEE nano-plate resting on Pasternak foundation based on nonlocal Mindlin theory. Pan and Waksmanski [28] presented exact closed-form solution for the 3D deformation of a layered MEE plate with nonlocal effect. Ansari and Gholami [29] studied the free vibration of MEE nano-plates in pre- and post-buckled states based on nonlocal Mindlin theory in conjunction with pseudo-arc length continuation approach. Farajpour et al. [30] used nonlocal Kirchhoff theory in conjunction with a perturbation method to obtain closed-form expression for nonlinear frequency of a MEE nano-plate with movable and immovable simply-supported boundary conditions. Ke et al. [31] presented a model based on nonlocal Love's shell theory to investigate the vibration response of a MEE cylindrical nano-shell. Ghadiri and Safarpour [32] investigated the free vibration response of MEE cylindrical nano-shells based on first-order shear deformation theory (FSDT) and Navier-type method. Mohammadimehr et al. [33] studied the free vibration of MEE cylindrical panels reinforced by carbon nano-tubes.

To the best of the author's knowledge, the free vibration of MEE doubly-curved nano-shells has not been studied. So, free vibration of a simply-supported magneto-electro-elastic doubly-curved nano-shell is studied based on FSDT and Gauss's laws for electrostatics and magneto statics. After obtaining a closed-form relation for the natural frequency, some examples are presented to validate the proposed method. Finally, the effects of several parameters on the natural frequencies of MEE doubly-curved nano-shells are investigated.

#### **2 PROBLEM MODELING**

Based on FSDT, the displacement field of a shallow doubly-curved shell is expressed by [34]:

$$
u = u_0 + z \theta_x
$$
,  $v = v_0 + z \theta_y$ ,  $w = w_0$  (1)

where  $u_0$ ,  $v_0$ , and  $w_0$  are the displacements of the mid-surface along *x*, *y*, and *z* directions, respectively, and  $\theta_x$  and  $\theta_y$ are the rotations of a transverse normal about the *y* and *x* directions, respectively. Using this displacement field, one

can obtain the strain–displacement relations [34]:  
\n
$$
\varepsilon_x = u_{0,x} + w_0/R_x + z \theta_{x,x}, \quad \varepsilon_y = v_{0,y} + w_0/R_y + z \theta_{y,y}
$$
\n(2)

$$
\varepsilon_x = u_{0,x} + w_0 / \Lambda_x + z \sigma_{x,x}, \quad \varepsilon_y = v_{0,y} + w_0 / \Lambda_y + z \sigma_{y,y}
$$
  
\n
$$
\gamma_{yz} = w_{0,y} + \theta_y, \quad \gamma_{xz} = w_{0,x} + \theta_x, \quad \gamma_{xy} = u_{0,y} + v_{0,x} + z \left( \theta_{x,y} + \theta_{y,x} \right)
$$
\n(3)

For a transversely-isotropic MEE nano-material, the constitutive relations can be written as [27]:

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\n
$$
\begin{pmatrix}\n\sigma_x \\
\sigma_y \\
\tau_{yz} \\
\tau_{zx} \\
\tau_{zy}\n\end{pmatrix} =\n\begin{bmatrix}\nC_{11} & C_{12} & 0 & 0 & 0 \\
C_{12} & C_{22} & 0 & 0 & 0 \\
0 & 0 & C_{44} & 0 & 0 \\
0 & 0 & 0 & C_{55} & 0 \\
0 & 0 & 0 & 0 & C_{66}\n\end{bmatrix}\n\begin{bmatrix}\n\varepsilon_x \\
\varepsilon_y \\
\varepsilon_x \\
\varepsilon_y \\
\varepsilon_x\n\end{bmatrix} +\n\begin{bmatrix}\n0 & 0 & e_{31} \\
0 & 0 & e_{32} \\
0 & e_{24} & 0 \\
0 & 0 & 0\n\end{bmatrix}\n\begin{bmatrix}\n0 \\
\varepsilon_z \\
\varepsilon_z \\
\varepsilon_z\n\end{bmatrix} +\n\begin{bmatrix}\n0 & 0 & 0 & 0 \\
0 & 0 & q_{32} \\
0 & q_{24} & 0 \\
0 & 0 & 0\n\end{bmatrix}\n\begin{bmatrix}\n\beta_{11} \\
\beta_{22} \\
\beta_{12} \\
\beta_{13} & 0 & 0 \\
0\n\end{bmatrix}
$$
\n
$$
\begin{pmatrix}\nT_{11} \\
T_{22} \\
T_{23} \\
T_{24}\n\end{pmatrix} =\n\begin{bmatrix}\n0 \\
0 \\
0 \\
0 \\
0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
0 \\
0 \\
0 \\
0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
0 \\
0 \\
0 \\
0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
0 \\
0 \\
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0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
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0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
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0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
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0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
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0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
0 \\
0 \\
0 \\
0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
0 \\
0 \\
0 \\
0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
0 \\
0 \\
0 \\
0\n\end{bmatrix} +\n\begin{bmatrix}\n0 \\
0 \\
0
$$

where  $\{\sigma\}$  and  $\{\epsilon\}$  are stress and strain vectors, respectively;  $\{\bm D\}$  and  $\{\bm B\}$  are the electric displacement and magnetic flux density vectors, respectively;  $\{E\} = \{0 \quad 0 \quad -\phi_{z}\}^{T}$  and  $\{H\} = \{0 \quad 0 \quad -\psi_{z}\}^{T}$  are electric field and magnetic field vectors, respectively, where *ϕ* and *ψ* denote electric and magnetic potentials; [*Cij*], [*ηij*] and [*μij*] are the elastic, dielectric and magnetic permeability coefficient matrices, respectively; [*eij*], [*qij*] and [*dij*] are the piezoelectric, piezo magnetic, and ME coefficient matrices, respectively;  $p_z$ ,  $m_z$  and  $\beta_{ii}$  are pyroelectric, pyro magnetic and thermal moduli, respectively;  $\Delta T$  denotes the temperature change,  $\nabla^2$  is the Laplace operator, and  $\eta$  is the nonlocal parameter revealing the size effect on the response of the nano-shell.



Using Hamilton's principle and based on FSDT, the equations of motion of the nano-shell resting on a Pasternak foundation (Fig.1) can be expressed by [34]:

$$
N_{x,x} + N_{xy,y} = 0 \tag{7}
$$

$$
N_{xy,x} + N_{y,y} = 0 \tag{8}
$$

$$
N_{xy,x} + N_{y,y} = 0
$$
\n
$$
Q_{x,x} + Q_{y,y} + (N_x w_{0,x} + N_{xy} w_{0,y})_x + (N_{xy} w_{0,x} + N_y w_{0,y})_y - \frac{N_x}{R_x} - \frac{N_y}{R_y} - k_w w_0 + k_s \nabla^2 w_0 = I_0 w_{0,t}
$$
\n(9)

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$$
M_{x,x} + M_{xy,y} - Q_x = I_2 \theta_{x,tt} \tag{10}
$$

$$
M_{xy,x} + M_{y,y} - Q_y = I_2 \theta_{y,t}
$$
 (11)

where  $\{N_x \mid N_y \mid N_{xy}\} = \int_{x}^{x} \sigma_x \sigma_y \sigma_{xy} d\sigma_y$ 2 2  $N_x$   $N_y$   $N_{xy}$   $=$   $\int_{-h/2}^{h/2} {\sigma_x \sigma_y \sigma_{xy}} dz$  are the in-plane force resultants,  ${Q_x Q_y}$   $=$   $K \int_{-h/2}^{h/2} {\sigma_{xz} \sigma_{yz}} dz$ 2 2  $\left\{Q_x \quad Q_y\right\} = K \int_{-h/2}^{h/2} \left\{\sigma_{xz} \quad \sigma_{yz}\right\} dz$ 

are the transverse force resultants,  $\{M_x \mid M_y \mid M_{xy}\} = \int_{x=0}^{h/2} \{\sigma_x \mid \sigma_y \mid \sigma_{xy}\}$ 2  $M_x$   $M_y$   $M_{xy}$  } =  $\int_{-h/2}^{h/2} {\sigma_x \sigma_y \sigma_{xy}} dz$  are the moments resultants, and

 $\{I_0 \mid I_2\} = \int_{-h/2}^{h/2} \{1 \mid z^2\} \rho_0$  $I_0$   $I_2$ } =  $\int_{-h/2}^{h/2}$  {1  $z^2$ } *ρ*<sub>0</sub>*dz* are the mass moments of inertia, in which *ρ*<sub>0</sub> is the material density and *K* is the shear correction factor. Moreover,  $k_w$  and  $k_s$  are spring and shear coefficients of the Pasternak foundation, respectively. To

determine theses resultants, the stress components given in Eq. (4) must be used. However, potentials *ϕ* and *ψ* are unknown parameters. To determine them, Gauss' laws for electrostatics and magneto statics are used:<br>  $D_{x,x} + D_{y,y} + D_{z,z} = 0$   $B_{x,x} + B_{y,y} + B_{z,z} = 0$ 

$$
D_{x,x} + D_{y,y} + D_{z,z} = 0 \qquad B_{x,x} + B_{y,y} + B_{z,z} = 0 \qquad (12)
$$

Then, using Eqs.  $(5)$ ,  $(6)$  and  $(12)$  gives:

$$
\phi_z = \phi_1 z + \phi_0 \tag{13}
$$

$$
\psi_{,z} = \psi_1 z + \psi_0 \tag{14}
$$

where  $\phi_0$  and  $\psi_0$  are constants of integration, and  $\phi_1$  and  $\psi_1$  are obtained by:

$$
M_{x,x} + M_{xy,y} - Q_x = I_2 Q_{y,x}
$$
\n(10)  
\n
$$
M_{xy,x} + M_{y,y} - Q_y = I_2 Q_{y,x}
$$
\n(11)  
\n
$$
\text{are } \{N_x - N_y - N_{xy}\} = \int_{-N/2}^{N/2} \{\sigma_x - \sigma_y\} \, dx \text{ the in-plane force resultants, } \{Q_x - Q_y\} = K \int_{-k/2}^{k/2} \{\sigma_x - \sigma_y\} \, dx
$$
\nthe transverse force results, 
$$
\{M_x - M_y - M_{xy}\} = \int_{-k/2}^{k/2} \{\sigma_x - \sigma_y\} \, dx \text{ are the moments resultants, and}
$$
\n
$$
I_2 = \int_{-k/2}^{k/2} \{\Gamma_x^2 + \frac{2}{3}\} \, \rho \, dx \text{ are the mass moments of inertia, in which } \rho_0 \text{ is the material density and } K \text{ is the shear number. Because values of the parameters of the parameters. To determine them, Gauss' layer in Eq. (4) must be used. However, potentials  $\phi$  and  $\psi$  are more than the mass results, the stress components given in Eq. (4) must be used. However, potentials  $\phi$  and  $\psi$  are more than the mass. To determine them, Gauss' laws for electrostatics and magnetic statistics are used:  
\n
$$
D_{x,x} + D_{y,y} + D_{z,z} = 0 \qquad B_{x,x} + B_{y,y} + B_{z,z} = 0 \qquad (12)
$$
\nThen, using Eqs. (5), (6) and (12) gives:  
\n
$$
\phi_x = \phi_x^2 + \phi_0
$$
\n
$$
\psi_x = \psi_x^2 + \psi_0
$$
\n
$$
\psi_x = \frac{1}{d_{x_1}^2 - \mu_{y_1} \eta_y} \{ (d_{x_1} q_{x_2} - \mu_{y_1} q_{x_2}) w_{x_1} + (d_{x_1} q_{x_2} - \mu_{y_1} q_{x_2}) w_{x_1} + [d_{x_1} (q_{x_1} + q_{x_1}) - \mu_{x_1} (q_{x_1} + q_{x_1})] \, \rho_{x_2} + [d_{x_1} (q_{x_1} + q_{x_1}) - \mu_{x_2} (q_{x_1} + q_{x_1})] \,
$$
$$

To determine *ϕ*<sup>0</sup> and *ψ*0, the ME boundary condition on top and bottom surfaces of the nano-shell is needed. If the nano-shell is poled along the *z* direction and subjected to electric potential  $V_0$  and magnetic potential  $\Omega_0$  between the upper and lower surfaces, the ME boundary condition can be expressed as below:

$$
\phi\left(x, y, z = -\frac{h}{2}\right) = 0, \quad \phi\left(x, y, z = +\frac{h}{2}\right) = V_0
$$
\n
$$
\psi\left(x, y, z = -\frac{h}{2}\right) = 0, \quad \psi\left(x, y, z = +\frac{h}{2}\right) = \Omega_0
$$
\n(16)

Eqs. (13) – (16) yield 
$$
\phi_0 = V_0/h
$$
 and  $\psi_0 = \Omega_0/h$ . Then, the resultants are obtained:  
\n
$$
(1 - \eta \nabla^2) N_x = C_{11} h u_{0,x} + C_{12} h v_{0,y} + \left(\frac{C_{11}}{R_x} + \frac{C_{12}}{R_y}\right) h w_0 + e_{31} V_0 + q_{31} \Omega_0 - \beta_{11} h \Delta T
$$
\n
$$
(1 - \eta \nabla^2) N_y = C_{12} h u_{0,x} + C_{22} h v_{0,y} + \left(\frac{C_{12}}{R_x} + \frac{C_{22}}{R_y}\right) h w_0 + e_{32} V_0 + q_{32} \Omega_0 - \beta_{22} h \Delta T
$$
\n(19)

$$
(17)
$$
\n
$$
\left(1-\eta \nabla^2\right) N_x = C_{11} n u_{0,x} + C_{12} n v_{0,y} + \left(\frac{C_{12}}{R_x} + \frac{C_{22}}{R_y}\right) n w_0 + e_{31} v_0 + q_{31} \Omega_0 - \beta_{11} n \Delta T \tag{17}
$$
\n
$$
\left(1-\eta \nabla^2\right) N_y = C_{12} n u_{0,x} + C_{22} n v_{0,y} + \left(\frac{C_{12}}{R_x} + \frac{C_{22}}{R_y}\right) n w_0 + e_{32} v_0 + q_{32} \Omega_0 - \beta_{22} n \Delta T \tag{18}
$$

$$
(1 - \eta \nabla^2) N_{xy} = C_{66} h \left( u_{0,y} + v_{0,x} \right) \tag{19}
$$

$$
(1 - \eta \nabla^2) M_x = M_1 w_{0,xx} + M_2 w_{0,yy} + M_3 \theta_{x,x} + M_4 \theta_{y,y}
$$
\n(20)

$$
(1 - \eta \nabla^2) M_y = M_1 w_{0,xx} + M_2 w_{0,yy} + M_5 \theta_{x,x} + M_6 \theta_{y,y}
$$
\n(21)

$$
(1 - \eta \nabla^2) M_{xy} = \frac{h^3}{12} C_{66} \left( \theta_{x,y} + \theta_{y,x} \right)
$$
 (22)

$$
(1 - \eta \nabla^2) Q_x = Kh C_{55} \left( w_{0,x} + \theta_x \right) \tag{23}
$$

$$
\left(1-\eta \nabla^2\right)Q_y = KhC_{44}\left(w_{0,y} + \theta_y\right) \tag{24}
$$

where

$$
M_{1} = \frac{e_{31}h^{3}(d_{33}q_{15} - \mu_{33}e_{15})}{12(d_{33}^{2} - \mu_{33}q_{33})} + \frac{q_{31}h^{3}(d_{33}e_{15} - \eta_{33}q_{15})}{12(d_{33}^{2} - \mu_{33}q_{33})}
$$
  
\n
$$
M_{2} = \frac{e_{31}h^{3}(d_{33}q_{24} - \mu_{33}e_{24})}{12(d_{33}^{2} - \mu_{33}q_{33})} + \frac{q_{31}h^{3}(d_{33}e_{24} - \eta_{33}q_{24})}{12(d_{33}^{2} - \mu_{33}q_{33})}
$$
  
\n
$$
M_{3} = \frac{h^{3}}{12}C_{11} + \frac{e_{31}h^{3}[d_{33}(q_{15} + q_{31}) - \mu_{33}(e_{15} + e_{31})]}{12(d_{33}^{2} - \mu_{33}q_{33})} + \frac{q_{31}h^{3}[d_{33}(e_{15} + e_{31}) - \eta_{33}(q_{15} + q_{31})]}{12(d_{33}^{2} - \mu_{33}q_{33})}
$$
  
\n
$$
M_{4} = \frac{h^{3}}{12}C_{12} + \frac{e_{31}h^{3}[d_{33}(q_{24} + q_{31}) - \mu_{33}(e_{24} + e_{31})]}{12(d_{33}^{2} - \mu_{33}q_{33})} + \frac{q_{31}h^{3}[d_{33}(e_{24} + e_{31}) - \eta_{33}(q_{24} + q_{31})]}{12(d_{33}^{2} - \mu_{33}q_{33})}
$$
  
\n
$$
M_{5} = M_{3} - \frac{h^{3}}{12}C_{11} + \frac{h^{3}}{12}C_{12}
$$
  
\n
$$
M_{6} = M_{4} - \frac{h^{3}}{12}C_{12} + \frac{h^{3}}{12}C_{22}
$$
  
\n
$$
(25)
$$

$$
(1-\eta \nabla^2) N_{xy} = C_{66} h (u_{0y} + v_{0x})
$$
\n
$$
(1-\eta \nabla^2) M_x = M_1 w_{0,xx} + M_2 w_{0,yy} + M_3 \theta_{x,x} + M_4 \theta_{y,y}
$$
\n
$$
(20)
$$
\n
$$
(1-\eta \nabla^2) M_y = M_1 w_{0,xx} + M_2 w_{0,yy} + M_5 \theta_{x,x} + M_6 \theta_{y,y}
$$
\n
$$
(1-\eta \nabla^2) M_{xy} = \frac{h^3}{12} C_{66} (\theta_{x,y} + \theta_{y,x})
$$
\n
$$
(1-\eta \nabla^2) Q_x = Kh C_{45} (w_{0x} + \theta_x)
$$
\n
$$
(1-\eta \nabla^2) Q_y = Kh C_{44} (w_{0y} + \theta_y)
$$
\nwhere\n
$$
M_1 = \frac{e_{31} h^3 (d_{33} q_{15} - \mu_{33} r_{33})}{12 (d_{33}^2 - \mu_{33} r_{33})} + \frac{q_{31} h^3 (d_{33} e_{15} - \eta_{33} q_{15})}{12 (d_{33}^2 - \mu_{33} r_{33})}
$$
\n
$$
M_2 = \frac{e_{31} h^3 (d_{33} q_{24} - \mu_{33} e_{34})}{12 (d_{33}^2 - \mu_{33} r_{33})} + \frac{q_{31} h^3 (d_{33} e_{24} - \mu_{33} q_{13})}{12 (d_{33}^2 - \mu_{33} r_{33})}
$$
\n
$$
M_3 = \frac{h^3}{12} C_{11} + \frac{e_{31} h^3 [d_{33} (q_{15} + q_{31}) - \mu_{33} (e_{15} + e_{31})]}{12 (d_{33}^2 - \mu_{33} r_{33})}
$$
\n
$$
M_4 = \frac{h^3}{12} C_{11} + \frac{e_{31} h^3 [d_{33} (q_{24} + q_{31}) - \mu_{33} (e_{15} + e_{31})]}{12 (d_{33}^2 - \mu_{33}
$$

The following ordinary differential equations are obtained:

$$
L_1 U + L_2 V + L_3 W = 0 \tag{27}
$$

$$
L_4 U + L_5 V + L_6 W = 0 \tag{28}
$$

$$
L_7\ddot{W} + L_8W + L_9U + L_{10}V + L_{11}X + L_{12}Y + L_{13} = 0
$$
\n(29)

$$
L_{14}\ddot{X} + L_{15}X + L_{16}Y + L_{17}W = 0
$$
\n(30)

$$
L_{18}\ddot{Y} + L_{19}Y + L_{20}X + L_{21}W = 0
$$
\n(31)

where *U*, *V*, *W*, *X* and *Y* are unknown functions and  $(m,n)$  denotes the mode of vibration. In Eqs. (27) – (31) the subscript '*mn*' has been dropped for brevity.

Eqs. (27) and (28) give *U* and *V* in terms of *W*, while Eqs. (30) and (31) give *X* and *Y*:

$$
U = \frac{L_3 L_5 - L_2 L_6}{L_2 L_4 - L_1 L_5} W , \qquad V = \frac{L_1 L_6 - L_3 L_4}{L_2 L_4 - L_1 L_5} W
$$
\n(32)

$$
X = \frac{L_{16}L_{21} - L_2^* L_{17}}{L_1^* L_2^* - L_{16} L_{20}} W, \quad Y = \frac{L_{17}L_{20} - L_1^* L_{21}}{L_1^* L_2^* - L_{16} L_{20}} W
$$
\n(33)

where  $L_{1}^{*} = L_{15} + L_{14} \frac{d^{2}}{dt^{2}}$  $L_1^* = L_{15} + L_{14} \frac{d}{dt}$ *dt*  $=L_{15}+L_{14}\frac{u}{2}$  and  $\frac{1}{2}$  = L<sub>19</sub> + L<sub>18</sub>  $\frac{d^2}{dt^2}$  $L_2^* = L_{19} + L_{18} \frac{d}{dt}$ *dt*  $=L_{19}+L_{18}\frac{a}{2}$  are ordinary differential operators.

Now, substituting Eqs. (32) and (33) into (29) gives:

$$
M_{eq}\ddot{W} + K_{eq}W + F_s = 0 \tag{34}
$$

In which  $M_{eq}$  and  $K_{eq}$  are equivalent mass and stiffness of the system and  $F_s$  is a constant parameter. These parameters are given in Appendix A. Finally, natural frequency of the nano-shell is simply determined by using  $\omega_0 = \sqrt{K_{\text{eq}}/M_{\text{eq}}}$ .

#### **3 RESULTS AND DISCUSSION**

In this section, first, some examples are presented to validate the proposed model. Then, the effects of several parameters on the natural frequencies of MEE doubly-curved nano-shells are investigated. The MEE nano-shell is doubly-curved and simply-supported. It is subjected to electric ( $V_0$ ) and magnetic (Ω<sub>0</sub>) potentials between its upper and bottom surfaces. For all the numerical examples the shear correction factor (*K*) is taken to be 5/6.

First, an isotropic spherical shell is considered and its fundamental natural frequency is obtained. The result is shown in Table 1., along with the published results based on Sanders theory (ST) [35], Donnell theory (DT) [36], HSDT [37] and 3D approach [38]. The geometric and material properties of the shell are:  $a = b = 1.0118$ ,  $h =$ 0.0191,  $R = 1.91$ ,  $E = 1$ ,  $\rho_0 = 1$  and  $v = 0.3$ . It is seen that the result obtained by present model is in good agreement with previously published ones, and the discrepancy between the results of 3D method and present method is very small.

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|-----------|--|
|           |  |

Fundamental natural frequency  $\omega_0$  (*rad/s*) of an isotropic spherical shell.



As the second comparison, the frequency ratio of an isotropic nano-plate with  $a = b = 10$  nm,  $h = 0.34$  nm,  $E =$ 1.06 *TPa*,  $\rho_0 = 2250 \text{ kg/m}^3$ , and  $v = 0.25$  is obtained for different values of nonlocal parameter. The results are

presented in Table 2., where frequency ratios are determined by using the following relation: frequencyratiocalculated using nonlocal theory Frequencyratio= frequencyratiocalculated usinglocal theory (35)

#### **Table 2**

Frequency ratios of a graphene sheet.



Table 3., presents the dimensionless fundamental frequencies of an isotropic nano-plate for different nonlocal parameters, and *a*/*h* and *b*/*a* ratios. The results are compared with the ones obtained by methods based on nonlocal Kirchhoff theory (CPT), FSDT, TSDT and a RPT. Again, it is seen that the proposed model predicts the frequencies with good accuracy.

### **Table 3**

Dimensionless fundamental frequency  $\omega = \omega_0 h \sqrt{\rho_0/G}$  of an isotropic Nano plate (*v* = 0.3).



reported by Malekzadeh and Shojaee [7]

Next, dimensionless fundamental frequencies of a MEE nano-plate made of piezoelectric BaTiO<sub>3</sub> and magnetostrictive  $\text{CoFe}_2\text{O}_4$  are determined. The material properties of the MEE material are given in Table 4. The dimensionless fundamental frequencies are obtained by using  $\omega = \omega_0 a \sqrt{I_0/(C_{11}h)}$ . Table 5., presents the frequencies for different values of nonlocal parameter. In addition, the effect of temperature rise on the dimensionless fundamental frequency has been investigated and the results are presented in Table 6. As in the case for isotropic shell and nano-plate, the proposed method approximates the frequencies of MEE nano-plate with good accuracy.





#### **Table 5**

The effect of nonlocal parameter (*η*) on the dimensionless fundamental frequency of a MEE nano-plate ( $\Delta T = V_0 = \Omega_0 = 0$ ).



#### **Table 6**

The effect of the temperature rise on the dimensionless fundamental frequency of a MEE nano-plate ( $\eta = 1.44 \times 10^{-16}$ ,  $V_0 = \Omega_0 =$ 0).



The effect of radius of curvature  $(R_x)$  on the dimensionless fundamental frequency of nanoscale spherical and cylindrical MEE shells is investigated and the result is shown in Fig. 2. In this example,  $a = b = 60$  nm,  $h = 4$  nm and  $\eta = 0.2$  *nm*<sup>2</sup>. It is seen that for a specific  $R_x$ , the spherical shell has higher frequency which decreases with higher rate when  $R_x$  increases.



#### **Fig.2**

Dimensionless fundamental frequency of spherical and cylindrical MEE nano-shells ( $\Delta T = V_0 = \Omega_0 = 0$ ).

Fig. 3 shows the effect of temperature change on the dimensionless fundamental frequency of two MEE nanoshells (with  $a = b = 60$  *nm*,  $h = 4$  *nm*, and  $R = 5a$ ) and a MEE nano-plate (with  $a = b = 60$  *nm*,  $h = 4$  *nm*). For all of the structures, the dimensionless frequencies decrease with increasing the temperature.

Effects of electric and magnetic potentials on the natural frequency of MEE nano-shells are also investigated and the results are shown in Figs. 4 and 5, respectively. The geometric properties of the nano-structures are the same as previous example. It is observed that increasing the electric potential, decreases the natural frequency of MEE nanoshells. Moreover, for negative electric potentials, higher natural frequencies are resulted. The converse happens when magnetic potential changes. That is, as the magnetic potential increases, the natural frequency of the MEE nano-shell increases.



#### **Fig.3**

Effect of temperature change on the dimensionless fundamental frequency of MEE nano-structures ( $\eta = 0.2$  nm<sup>2</sup>,  $V_0 = \Omega_0 = 0$ ).

#### **Fig.4**

Effect of electric potential on the dimensionless fundamental frequency of MEE nano-structures ( $\eta = 0.2$  nm<sup>2</sup>,  $\Delta T = \Omega_0 = 0$ ).

#### **Fig.5**

Effect of magnetic potential on the dimensionless fundamental frequency of MEE nano-structures ( $\eta = 0.2$  nm<sup>2</sup>,  $\Delta T = V_0 = 0$ ).

Effects of nonlocal parameter and foundation parameters on the natural frequencies of MEE doubly-curved nano-shells are investigated, too. Fig. 6 shows the effect of nonlocal parameter on the dimensionless fundamental frequencies of spherical and cylindrical MEE nano-shells with  $a = b = 60$  nm,  $h = 4$  nm,  $R = 5a$ . It is seen that nonlocal parameter has small effect on the frequency and tends to decrease it. In Fig. 7, effects of foundation parameters on the dimensionless fundamental frequency of spherical MEE nano-shell with  $a = b = 60$  nm,  $h = 4$  nm,  $R = 5a$  are shown. The dimensionless foundation parameters are obtained by  $K_w = k_w a^4 / (C_{ijmax} h^3)$  and  $K_s =$  $k_s a^2/(C_{ijmax}h^3)$  where  $C_{ijmax}$  is the maximum value of the stiffness coefficients of the MEE nano-shell. It is observed that both dimensionless spring and shear coefficients increase the stiffness (and subsequently the natural frequency) of the system. However, dimensionless shear coefficient  $(K<sub>s</sub>)$  has more effect on the natural frequency of MEE nanoshells.

**Fig.6**



# Variation of dimensionless fundamental frequency of MEE nano-shell in terms of nonlocal parameter ( $\Delta T = V_0 = \Omega_0 = 0$ ).



Variation of dimensionless fundamental frequency of spherical MEE nano-shell in terms of foundation parameters  $(\eta = 0.2 \text{ nm}^2, \Delta T = V_0 = \Omega_0 = 0).$ 

# **4 CONCLUSION**S

Free vibration of simply-supported MEE doubly-curved nano-shells is studied analytically based on FSDT. Some examples are presented and it is found that: (a) spherical nano-shell has higher natural frequencies which decrease with higher rate when the curvature of the nano-shell increases, (b) natural frequency decreases with increasing the temperature, (c) increasing the electric potential decreases the natural frequency of MEE nano-shells, (d) when the magnetic potential increases, the natural frequency of the MEE nano-shell increases, (e) nonlocal parameter decreases the natural frequency of MEE nano-shell, and (f) foundation parameters increase the natural frequencies of MEE nano-shells. However, dimensionless shear coefficient has more effect on the natural frequencies.

**Fig.7** 

# **APPENDIX A**

The coefficients of Eq. (34) are:

$$
M_{\text{eq}} = \alpha_5 L_7 + \alpha_6 L_8 + \alpha_6 (k_1 + k_2) - \alpha_2 L_{11} - \alpha_4 L_{12}
$$
\n(A.1)

$$
K_{\text{eq}} = \alpha_5 L_3 + \alpha_6 (k_1 + k_2) + L_{11} (L_{16} L_{21} - \alpha_1) + L_{12} (L_{17} L_{20} - \alpha_3)
$$
\n(A.2)

$$
F_s = \alpha_{15} L_{13} \tag{A.3}
$$

Where

$$
\alpha_1 = L_{17}L_{19}, \quad \alpha_2 = L_{17}L_{18}, \quad \alpha_3 = L_{15}L_{21}, \quad \alpha_1 = L_{14}L_{21}, \quad \alpha_5 = L_{15}L_{19} - L_{16}L_{20}, \quad \alpha_6 = L_{14}L_{19} - L_{16}L_{20}
$$
(A.4)

$$
k_1 = \frac{L_9 (L_3 L_5 - L_2 L_6)}{L_2 L_4 - L_1 L_5}, \qquad k_2 = \frac{L_{10} (L_1 L_6 - L_3 L_4)}{L_2 L_4 - L_1 L_5}
$$
(A.5)

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